

Lecture # 09: Conservation Equations of Mass and Momentum in Differential Form-P1

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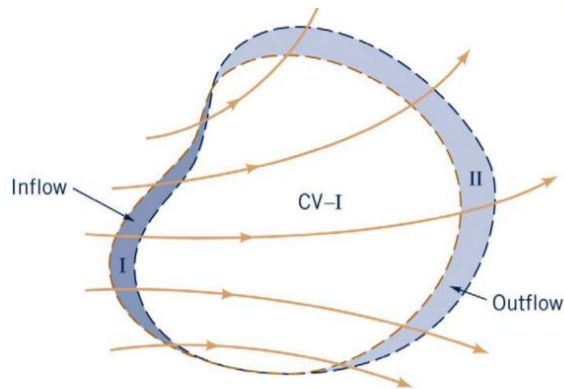
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Reynolds Transport Theorem

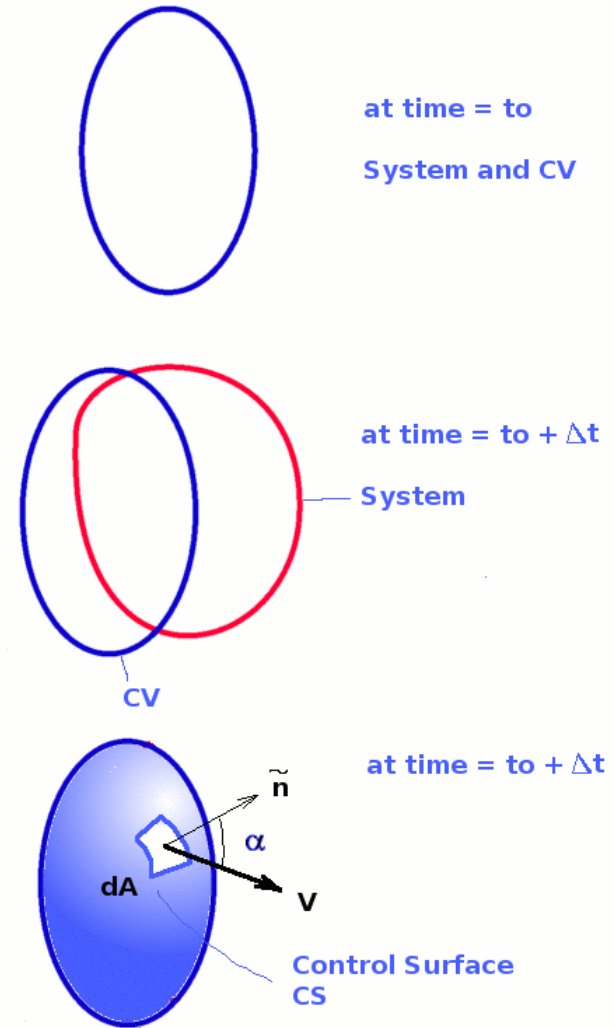
$$\frac{DN_s}{Dt} = \frac{D \int_V \alpha \rho dV}{Dt} = \frac{\partial}{\partial t} \int_{C.V.} \alpha \rho dV + \int_{C.S.} (\alpha \rho \vec{V}) \cdot d\vec{A}$$

Where α is any intensive property corresponding to N . (i.e., $\alpha = N$ per unit mass) and it can be used for different quantities as follows.

N_s	α
Mass	1
Linear momentum	\vec{V}
Angular momentum	$\vec{R} \times \vec{V}$
Energy	e
Entropy	s



--- Fixed control surface and system boundary at time t
 --- System boundary at time $t + \delta t$



at time = t_0
 System and CV

at time = $t_0 + \Delta t$
 System

at time = $t_0 + \Delta t$
 Control Surface CS

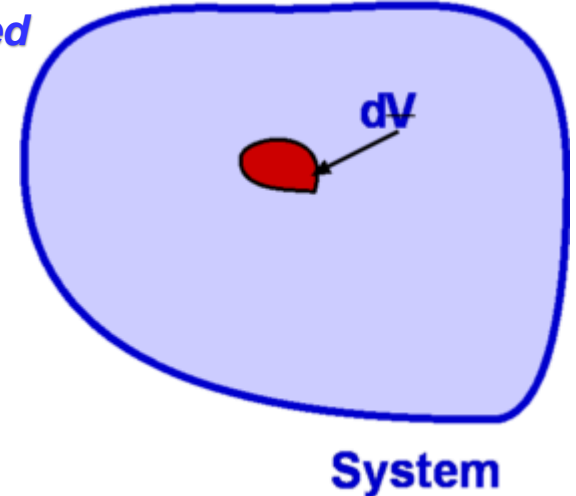
□ Conservation of Mass

Physical principle: Mass can be neither created or destroyed

$$m_{system} = constant$$

$$\left(\frac{dm}{dt}\right)_{system} = 0$$

$$with \quad m_{system} = \int_{system} dm = \int_{system} \rho dV$$

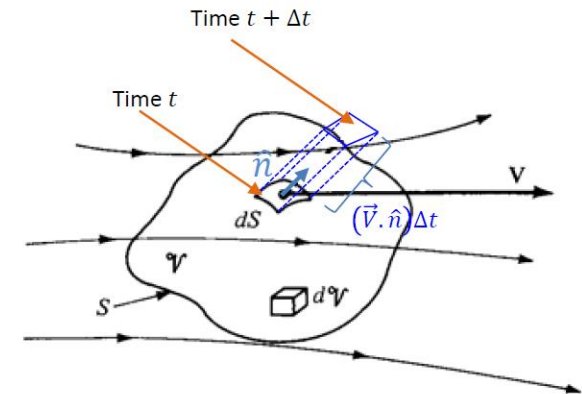


$$\frac{DN_s}{Dt} = \frac{D \int_V \alpha \rho dV}{Dt} = \frac{\partial}{\partial t} \int_{C.V.} \alpha \rho dV + \int_{C.S.} (\alpha \rho \vec{V}) \cdot d\vec{A}$$

$$Make: \quad \alpha = 1 \quad \frac{DM_s}{Dt} = \frac{\partial}{\partial t} \int_{C.V.} \rho dV + \int_{C.S.} (\rho \vec{V}) \cdot d\vec{A} = 0$$

Integral form of the Mass Conservation Equation:

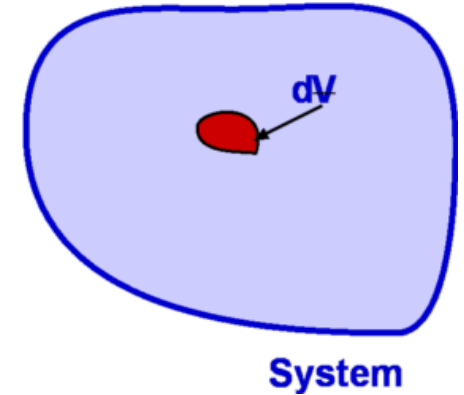
$$\frac{\partial}{\partial t} \int_{C.V.} \rho dV + \int_{C.S.} (\rho \vec{V}) \cdot d\vec{A} = 0$$



□ Conservation of Mass

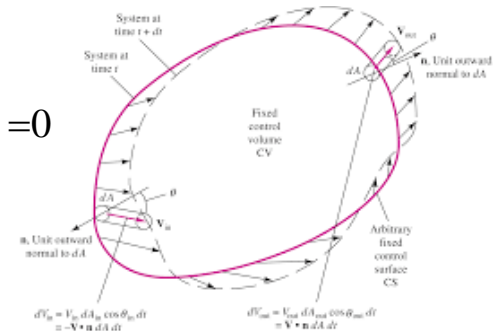
- Physical principle: Mass can be neither created or destroyed.

- Integral form:
$$\int_{C.V.} \frac{\partial \rho}{\partial t} dV + \int_{C.S.} (\rho \vec{V}) \cdot d\vec{A} = 0$$



- Applying **Guess divergence theorem**, we convert the surface integral to volume integral to obtain:

$$\int_{C.V.} \frac{\partial \rho}{\partial t} dV + \int_{C.S.} (\rho \vec{V}) \cdot d\vec{A} = \int_{C.V.} \frac{\partial \rho}{\partial t} dV + \int_{C.V.} \nabla \cdot (\rho \vec{V}) dV = \int_{C.V.} \left[\frac{\partial \rho}{\partial t} + \nabla \cdot (\rho \vec{V}) \right] dV = 0$$



- Differential form of the mass conservation equation (or continuity equation):

$$\begin{aligned} \frac{\partial \rho}{\partial t} + \nabla \cdot (\rho \vec{V}) &= \frac{\partial \rho}{\partial t} + \vec{V} \cdot \nabla \rho + \rho \nabla \cdot \vec{V} \\ &= \frac{D\rho}{Dt} + \rho \nabla \cdot \vec{V} = 0 \end{aligned}$$

Simplifications:

Form incompressible flows:

ρ is constant, then $\frac{\partial \rho}{\partial t} = 0$; $\nabla \rho = 0$

Therefore, $\nabla \cdot \vec{V} = 0$

□ Conservation of Mass

Example 01:

- *The x-component velocity is given by $u(x,y)=Ay^2+C$ in an 2D incompressible flow.*
 - *Please determine y-component velocity $v(x,y)$ if $v(x,0)=0$ as would be the case in flow between parallel plates.*

□ Conservation of Mass

Solution to Example 01:

$$\frac{D\rho}{Dt} + \rho \nabla \cdot \vec{V} = 0$$

$$\Rightarrow \nabla \cdot \vec{V} = 0$$

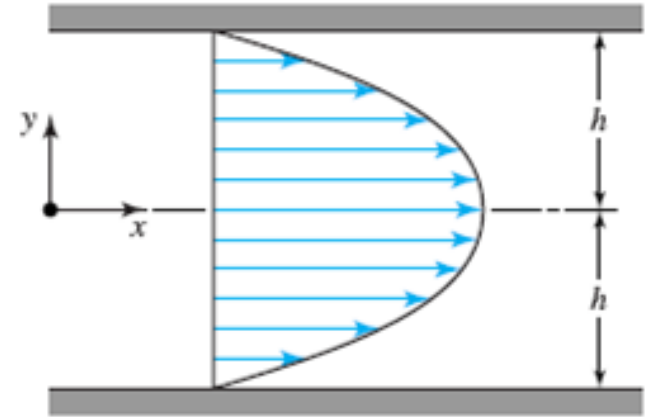
$$\Rightarrow \frac{\partial u}{\partial x} + \frac{\partial v}{\partial y} = 0$$

$$\Rightarrow \frac{\partial v}{\partial y} = -\frac{\partial u}{\partial x} = -\frac{\partial(Ay^2)}{\partial x} = 0$$

$$\frac{\partial v}{\partial y} = 0 \Rightarrow v(x, y) = f(x)$$

$$\text{But } \Rightarrow v(x, 0) = 0 \Rightarrow f(x) = 0$$

$$\Rightarrow v(x, y) = 0$$



(B)

$$u(y) = u_{\max} \left[1 - \left(\frac{y}{h} \right)^2 \right]$$

□ Conservation of Mass

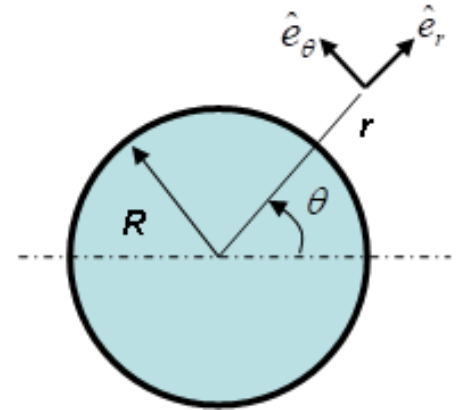
Example 02:

For a two-dimensional steady, inviscid, incompressible flow around a cylinder of radius of R as shown in the figure, the velocity field is given as :

$$\vec{V}(r, \theta) = U_\infty \left(1 - \frac{R^2}{r^2}\right) \cos \theta \cdot \hat{e}_r - U_\infty \left(1 + \frac{R^2}{r^2}\right) \sin \theta \cdot \hat{e}_\theta;$$

where U_∞ is the velocity of the undisturbed stream (therefore, U_∞ is constant).

- *Is the flow with the velocity field given above physically possible?*



$$\begin{aligned} \frac{\partial \rho}{\partial t} + \nabla \cdot (\rho \vec{V}) &= \frac{\partial \rho}{\partial t} + \vec{V} \cdot \nabla \rho + \rho \nabla \cdot \vec{V} \\ &= \frac{D\rho}{Dt} + \rho \nabla \cdot \vec{V} = 0 \end{aligned}$$

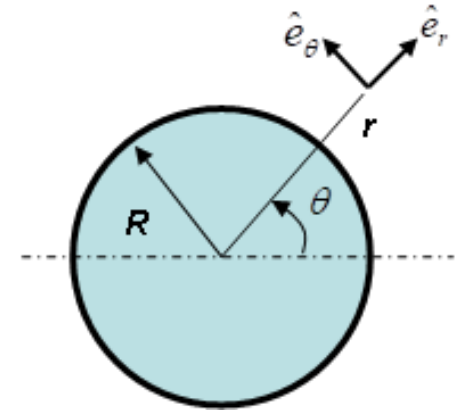
$$\nabla \cdot \vec{V} = \frac{1}{h_1 h_2 h_3} \left[\frac{\partial (h_2 h_3 V_1)}{\partial q_1} + \frac{\partial (h_1 h_3 V_2)}{\partial q_2} + \frac{\partial (h_1 h_2 V_3)}{\partial q_3} \right]$$

□ Conservation of Mass

For a two-dimensional steady, inviscid, incompressible flow around a cylinder of radius of R as shown in the figure, the velocity field is given as :

$$\vec{V}(r, \theta) = U_{\infty} \left(1 - \frac{R^2}{r^2}\right) \cos \theta \cdot \hat{e}_r - U_{\infty} \left(1 + \frac{R^2}{r^2}\right) \sin \theta \cdot \hat{e}_{\theta};$$

where U_{∞} is the velocity of the undisturbed stream (therefore, U_{∞} is constant).



- *Is the flow with the velocity field given above physically possible?*

For a steady, inviscid, incompressible flow, the mass conservation equation will be: $\nabla \cdot \vec{V} = 0$
 Expend it in the cylindrical system, it will be:

$$\frac{1}{r} \left[\frac{\partial(rV_r)}{\partial r} + \frac{\partial V_{\theta}}{\partial \theta} + \frac{\partial(rV_z)}{\partial z} \right] = 0 \quad \Rightarrow \quad \frac{\partial(rV_r)}{\partial r} + \frac{\partial V_{\theta}}{\partial \theta} + \frac{\partial(rV_z)}{\partial z} = 0$$

For the velocity field given above:

$$V_r = U_{\infty} \left(1 - \frac{R^2}{r^2}\right) \cos \theta; \quad V_{\theta} = -U_{\infty} \left(1 + \frac{R^2}{r^2}\right) \sin \theta; \quad V_z = 0$$

Since

$$\frac{\partial(rV_r)}{\partial r} = \frac{\partial}{\partial r} \left[U_{\infty} \left(r - \frac{R^2}{r} \right) \cos \theta \right] = U_{\infty} \left(1 + \frac{R^2}{r^2} \right) \cos \theta$$

$$\frac{\partial V_{\theta}}{\partial \theta} = \frac{\partial}{\partial \theta} \left[-U_{\infty} \left(1 + \frac{R^2}{r^2} \right) \sin \theta \right] = -U_{\infty} \left(1 + \frac{R^2}{r^2} \right) \cos \theta$$

$$\frac{\partial(rV_z)}{\partial z} = 0$$

Therefore:

$$\frac{\partial(rV_r)}{\partial r} + \frac{\partial V_{\theta}}{\partial \theta} + \frac{\partial(rV_z)}{\partial z} = U_{\infty} \left(1 + \frac{R^2}{r^2} \right) \cos \theta - U_{\infty} \left(1 + \frac{R^2}{r^2} \right) \cos \theta + 0 = 0$$

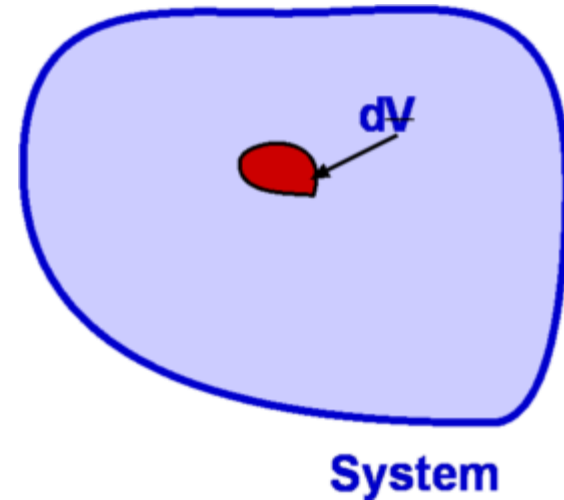
The velocity satisfies the mass conservation equation, i. e., with the velocity field given above is physically possible!

$$\nabla \cdot \vec{V} = \frac{1}{h_1 h_2 h_3} \left[\frac{\partial(h_2 h_3 V_1)}{\partial q_1} + \frac{\partial(h_1 h_3 V_2)}{\partial q_2} + \frac{\partial(h_1 h_2 V_3)}{\partial q_3} \right]$$

□ Conservation of Momentum

- Newton's second law states that:
[Time change rate of momentum of a system] = [Resultant external force acting on the system]

$$\frac{d\vec{M}_s}{dt} = \sum \vec{F}_s = \sum \vec{F}_{Surface} + \sum \vec{F}_{body}$$



Reynolds Transport Theorem:

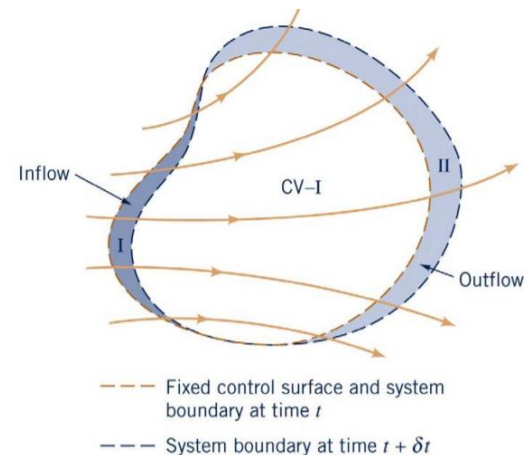
$$\left. \frac{dN_s}{dt} \right|_{system} = \frac{d \int_V \alpha \rho dV}{Dt} = \frac{\partial}{\partial t} \int_{C.V.} \alpha \rho dV + \int_{C.S.} (\alpha \rho \vec{V}) \cdot d\vec{A}$$

Make: $\alpha = \vec{V}$

$$\left. \frac{DM_s}{Dt} \right|_{system} = \frac{\partial}{\partial t} \int_{C.V.} \rho \vec{V} dV + \int_{C.S.} (\vec{V} \rho \vec{V}) \cdot d\vec{A} = 0$$

- Integral form of the Mass Conservation Equation:**

$$\frac{\partial}{\partial t} \int_{C.V.} \vec{V} \rho dV + \int_{C.S.} (\vec{V} \rho \vec{V}) \cdot d\vec{A} = \vec{F}_{surface} + \vec{F}_{body}$$



□ Conservation of Momentum

$$\frac{\partial}{\partial t} \int_{C.V.} \vec{V} \rho \, dV + \int_{C.S.} (\vec{V} \rho \vec{V}) \cdot d\vec{A} = \vec{F}_{surface} + \vec{F}_{body}$$

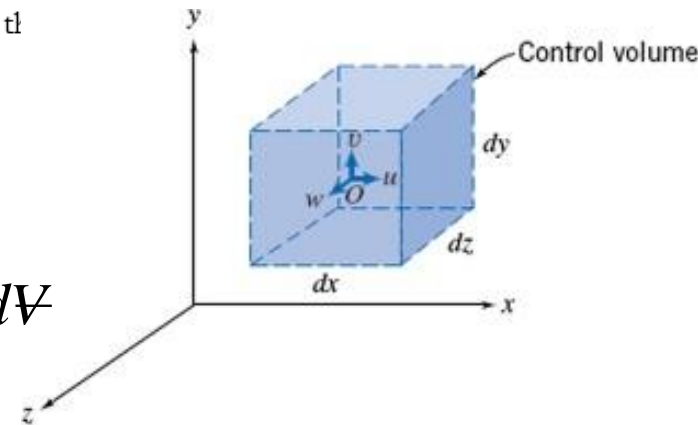
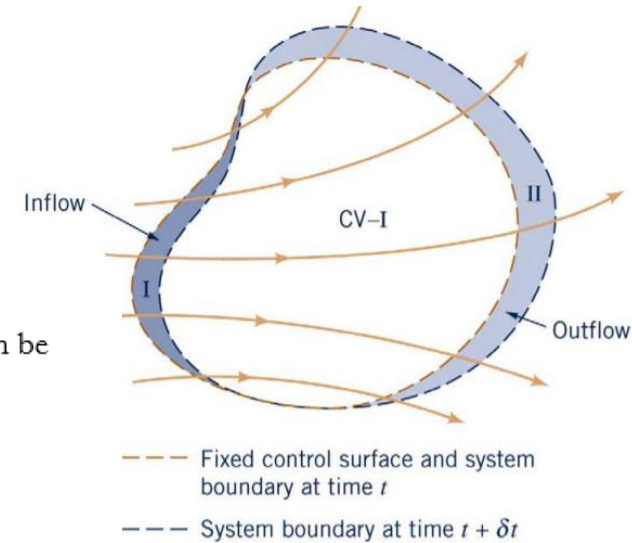
$\vec{F}_{surface}$

Surface forces such as pressure and shear stress. The surface forces usually can be expressed as $\vec{F}_{surface} = \int_{C.S.} \tilde{P} \cdot d\vec{A}$, where \tilde{P} is the stress tensor exerted by the surroundings on the particle surface. $\tilde{P} = -P\vec{I} + \tilde{\tau}$

\vec{F}_{body}

Body forces such as electromagnetic, gravitational forces. Usually the body force can be expressed as $\vec{F}_{body} = \int_{C.V.} \rho \vec{f} \, dV$, where \vec{f} is a vector which references the resultant force per unit mass.

$$\frac{\partial}{\partial t} \int_{C.V.} \rho \vec{V} \, dV + \int_{C.S.} (\rho \vec{V} \vec{V}) \cdot d\vec{A} = \int_{C.S.} \tilde{P} \cdot d\vec{A} + \int_{C.V.} \rho \vec{f} \, dV$$



□ Conservation of Momentum

- **Using divergence theorem for the control surface integrals, we obtained following equation after noting that the limits do not change.**

$$\frac{\partial}{\partial t} \int_{c.v.} \rho \vec{V} dV + \int_{c.s.} \nabla \cdot (\rho \vec{V} \vec{V}) dV = \int_{c.s.} \nabla \cdot \tilde{P} dV + \int_{c.v.} \rho \vec{f} dV$$

$$\Rightarrow \int_{c.v.} \left[\frac{\partial(\rho \vec{V})}{\partial t} + \nabla \cdot (\rho \vec{V} \vec{V}) - \nabla \cdot \tilde{P} - \rho \vec{f} \right] dV = 0$$

$$\Rightarrow \frac{\partial(\rho \vec{V})}{\partial t} + \nabla \cdot (\rho \vec{V} \vec{V}) - \nabla \cdot \tilde{P} - \rho \vec{f} = 0$$

- **Expand the above equation using**

$$\nabla \cdot (\phi \vec{A}) = (\vec{A} \cdot \nabla) \phi + \phi \nabla \cdot \vec{A}$$

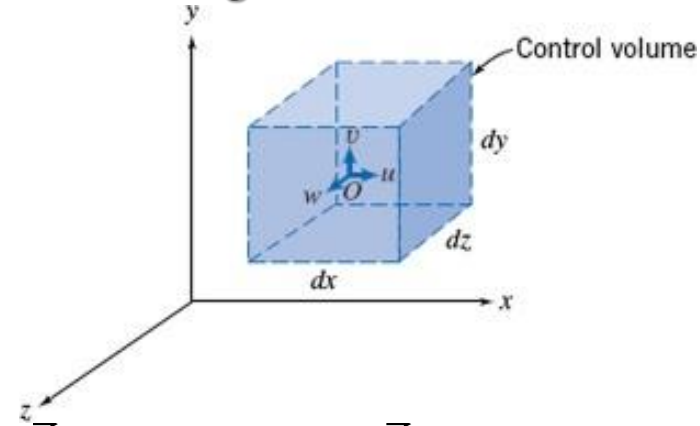
$$\frac{\partial(\rho \vec{V})}{\partial t} + \nabla \cdot (\rho \vec{V} \vec{V}) - \nabla \cdot \tilde{P} - \rho \vec{f} = 0$$

$$\Rightarrow \vec{V} \frac{\partial \rho}{\partial t} + \rho \frac{\partial \vec{V}}{\partial t} + \vec{V} \nabla \cdot (\rho \vec{V}) + (\rho \vec{V} \cdot \nabla) \vec{V} - \nabla \cdot \tilde{P} - \rho \vec{f} = 0$$

$$\Rightarrow \vec{V} \left[\frac{\partial \rho}{\partial t} + \nabla \cdot (\rho \vec{V}) \right] + \rho \frac{\partial \vec{V}}{\partial t} + (\rho \vec{V} \cdot \nabla) \vec{V} - \nabla \cdot \tilde{P} - \rho \vec{f} = 0$$

$$\Rightarrow \rho \frac{\partial \vec{V}}{\partial t} + (\rho \vec{V} \cdot \nabla) \vec{V} - \nabla \cdot \tilde{P} - \rho \vec{f} = 0$$

$$\Rightarrow \rho \frac{D\vec{V}}{Dt} - \nabla \cdot \tilde{P} - \rho \vec{f} = 0$$



- **The differential form of the momentum equation is:**

$$\rho \frac{D\vec{V}}{Dt} - \nabla \cdot \tilde{P} - \rho \vec{f} = 0$$

□ The Navier-Stokes Equations

$$\frac{\partial(\rho\vec{V})}{\partial t} + \nabla \cdot (\rho\vec{V}\vec{V}) - \nabla \cdot \tilde{P} - \rho\vec{f} = 0$$

Where $\tilde{P} = -P\tilde{I} + \tilde{\tau}$, and tensor $\tilde{I} = \begin{vmatrix} 1 & 0 & 0 \\ 0 & 1 & 0 \\ 0 & 0 & 1 \end{vmatrix}$; and $\tilde{\tau} = \begin{vmatrix} \sigma_{xx} & \tau_{yx} & \tau_{zx} \\ \tau_{xy} & \sigma_{yy} & \tau_{zy} \\ \tau_{xz} & \tau_{yz} & \sigma_{zz} \end{vmatrix}$

Re-writing the equation after substitution leads to:

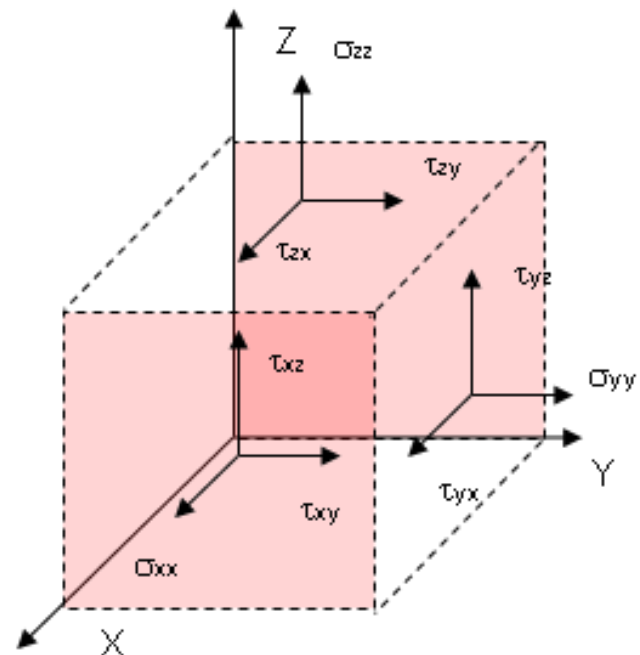
$$\frac{\partial(\rho\vec{V})}{\partial t} + \nabla \cdot (\rho\vec{V}\vec{V}) - \nabla \cdot (-P\tilde{I} + \tilde{\tau}) - \rho\vec{f} = 0$$

$$\frac{\partial(\rho\vec{V})}{\partial t} + \nabla \cdot (\rho\vec{V}\vec{V} + P\tilde{I} - \tilde{\tau}) - \rho\vec{f} = 0$$

Since $\nabla \cdot (P\tilde{I}) = P\nabla \cdot \tilde{I} + (\tilde{I} \cdot \nabla)P = \nabla P$

Therefore:

$$\frac{\partial(\rho\vec{V})}{\partial t} + \nabla \cdot (\rho\vec{V}\vec{V}) + \nabla P - \nabla \cdot \tilde{\tau} - \rho\vec{f} = 0$$



□ The Navier-Stokes Equations

Another form of Navier-Stokes equation:

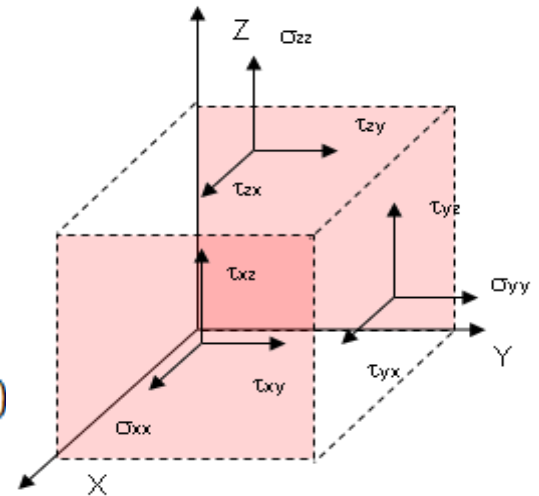
$$\frac{\partial(\rho \vec{V})}{\partial t} + \nabla \cdot (\rho \vec{V} \vec{V}) + \nabla P - \nabla \cdot \tilde{\tau} - \rho \vec{f} = 0$$

$$\Rightarrow \rho \frac{\partial \vec{V}}{\partial t} + \vec{V} \frac{\partial \rho}{\partial t} + \vec{V} \nabla \cdot (\rho \vec{V}) + (\rho \vec{V} \cdot \nabla) \vec{V} + \nabla P - \nabla \cdot \tilde{\tau} - \rho \vec{f} = 0$$

$$\Rightarrow \vec{V} \underbrace{\left[\frac{\partial \rho}{\partial t} + \nabla \cdot (\rho \vec{V}) \right]}_{\text{continuity equation}} + \rho \frac{\partial \vec{V}}{\partial t} + (\rho \vec{V} \cdot \nabla) \vec{V} + \nabla P - \nabla \cdot \tilde{\tau} - \rho \vec{f} = 0$$

$$\Rightarrow \rho \left(\underbrace{\frac{\partial \vec{V}}{\partial t} + \vec{V} \cdot \nabla \vec{V}}_{\text{substantial derivative}} \right) + \nabla P - \nabla \cdot \tilde{\tau} - \rho \vec{f} = 0 \Rightarrow \rho \frac{D\vec{V}}{Dt} + \nabla P - \nabla \cdot \tilde{\tau} - \rho \vec{f} = 0$$

$$\text{i.e., } \rho \frac{D\vec{V}}{Dt} + \nabla P - \nabla \cdot \tilde{\tau} - \rho \vec{f} = 0$$



□ The Navier-Stokes Equations

$$\rho \frac{D\vec{V}}{Dt} + \nabla P - \nabla \cdot \tilde{\tau} - \rho \vec{f} = 0$$

Stress Tensor

The stress tensor has nine components:

$$\tilde{\tau} = \begin{bmatrix} \sigma_{xx} & \tau_{yx} & \tau_{zx} \\ \tau_{xy} & \sigma_{yy} & \tau_{zy} \\ \tau_{xz} & \tau_{yz} & \sigma_{zz} \end{bmatrix}$$

Newtonian fluid,

$$\tilde{\tau} = \mu[\nabla\vec{V} + (\nabla\vec{V})^T - \frac{2}{3}(\nabla \cdot \vec{V})\tilde{I}]$$

For incompressible flow, in Cartesian coordinate system

$$\tau_{ij} = \mu \left(\frac{\partial u_i}{\partial x_j} + \frac{\partial u_j}{\partial x_i} \right)$$

$$\tau_{xy} = \tau_{yx}; \quad \tau_{xz} = \tau_{zx} \quad \tau_{zy} = \tau_{yz}$$

