# Lecture # 10: Conservation Equations of Mass and Momentum in Differential Form-P2

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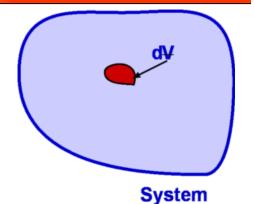


## Conservation of Mass

Physical principle: Mass can be neither created or destroyed.

• Integral form:

$$\int_{C.\Psi} \frac{\partial \rho}{\partial t} \, d\Psi + \int_{C.S.} (\rho \vec{V}) \bullet d\vec{A} = 0$$



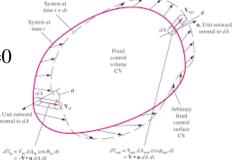
 Applying Guess divergence theorem, we convert the surface integral to volume integral to obtain:

$$\int_{C.\Psi.} \frac{\partial \rho}{\partial t} \, d\Psi + \int_{C.S.} (\rho \vec{V}) \bullet d\vec{A} = \int_{C.V.} \frac{\partial \rho}{\partial t} \, d\Psi + \int_{C.V.} \nabla \bullet (\rho \vec{V}) \, d\Psi = \int_{C.V.} \left[ \frac{\partial \rho}{\partial t} + \nabla \bullet (\rho \vec{V}) \right] d\Psi = 0$$

• Differential form of the mass conservation equation (or continuity equation):

$$\frac{\partial \rho}{\partial t} + \nabla \bullet (\rho \vec{V}) = \frac{\partial \rho}{\partial t} + \vec{V} \bullet \nabla \rho + \rho \nabla \bullet \vec{V}$$
$$= \frac{D\rho}{Dt} + \rho \nabla \bullet \vec{V} = 0$$

Simplifications: Form incompressible flows:  $\rho$  is constant, then  $\frac{\partial \rho}{\partial t} = 0;$   $\nabla \rho = 0$ Therefore,  $\nabla \bullet \vec{V} = 0$ 



## Conservation of Momentum

Newton's second law states that:

[Time change rate of momentum of a system] = [Resultant external force acting on the system]

$$\frac{d\vec{M}_{S}}{dt} = \sum \vec{F}_{S} = \sum \vec{F}_{Surface} + \sum \vec{F}_{body}$$

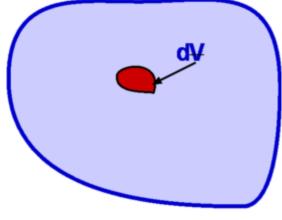
#### **Reynolds Transport Theorem:**

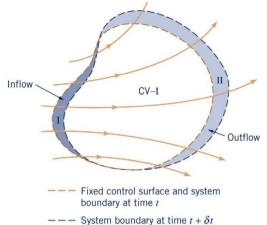
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$$\frac{dN_s}{dt}\Big|_{system} = \frac{d\int \alpha \rho \, d\Psi}{Dt} = \frac{\partial}{\partial t} \int_{C.V.} \alpha \rho \, d\Psi + \int_{C.S.} (\alpha \rho \vec{V}) \bullet d\vec{A}$$
  
Make:  $\alpha = \vec{V} \quad \frac{DM_s}{Dt}\Big|_{system} = \frac{\partial}{\partial t} \int_{C.V.} \rho \vec{V} \, d\Psi + \int_{C.S.} (\vec{V} \rho \vec{V}) \bullet d\vec{A} = 0$ 

Integral form of the Mass Conservation Equation:

$$\frac{\partial}{\partial t} \int_{C.V.} \vec{V} \rho \, d\Psi + \int_{C.S.} (\vec{V} \rho \vec{V}) \bullet d\vec{A} = \vec{F}_{surface} + \vec{F}_{body}$$





### The Navier-Stokes Equations

$$\rho \frac{D\vec{V}}{Dt} + \nabla P - \nabla \bullet \tilde{\tau} - \rho \, \vec{f} = 0$$

 $\tau_{yz}$ 

#### Stress Tensor

The stress tensor has nine components:

$$\widetilde{\tau} = \begin{vmatrix} \sigma_{xx} & \tau_{yx} & \tau_{zx} \\ \tau_{xy} & \sigma_{yy} & \tau_{zy} \\ \tau_{xz} & \tau_{yz} & \sigma_{zz} \end{vmatrix}$$

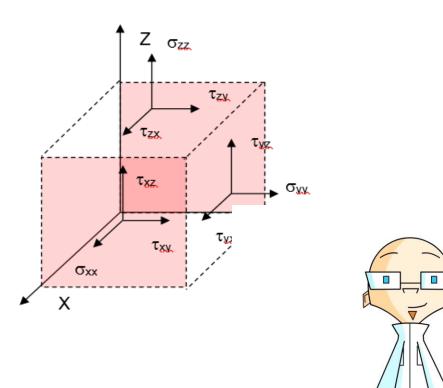
Newtonian fluid,

$$\widetilde{\tau} = \mu [\nabla \vec{V} + (\nabla \vec{V})^T - \frac{2}{3} (\nabla \bullet \vec{V}) \widetilde{I}]$$

For incompressible <u>flow, in</u> Cartesian coordinate system

$$\tau_{ij} = \mu \left( \frac{\partial u_i}{\partial x_j} + \frac{\partial u_j}{\partial x_i} \right)$$

$$\tau_{xy} = \tau_{yx}; \qquad \tau_{xz} = \tau_{zx} \qquad \tau_{zy} =$$



• Newtonian and non-Newtonian fluids

### **THE NAVIER-STOKES EQUATION IN CARTESIAN COORDINATE**

$$\rho \frac{D\vec{V}}{Dt} + \nabla P - \nabla \bullet \tilde{\tau} - \rho \, \vec{f} = 0$$

For incompressible flow, Navier-Stokes equation in Cartesian coordinate system will be:

 $\left[\rho \frac{Du}{Dt} = -\frac{\partial p}{\partial x} + \rho f_x + \mu \left[\frac{\partial^2 u}{\partial x^2} + \frac{\partial^2 u}{\partial y^2} + \frac{\partial^2 u}{\partial z^2}\right]\right]$ 

 $\int \rho \frac{Dv}{Dt} = -\frac{\partial p}{\partial y} + \rho f_y + \mu \left[\frac{\partial^2 v}{\partial x^2} + \frac{\partial^2 v}{\partial y^2} + \frac{\partial^2 v}{\partial z^2}\right]$ 

x-direction :

$$\int_{0}^{0} \rho \left[\frac{\partial u}{\partial t} + u\frac{\partial u}{\partial x} + v\frac{\partial u}{\partial y} + w\frac{\partial u}{\partial z}\right] = -\frac{\partial p}{\partial x} + \rho f_x + \mu \left[\frac{\partial^2 u}{\partial x^2} + \frac{\partial^2 u}{\partial y^2} + \frac{\partial^2 u}{\partial z^2}\right]$$

y-direction:

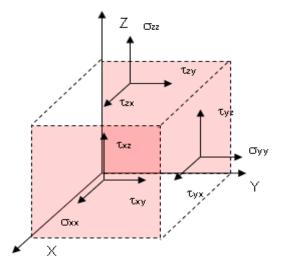
$$\left[\rho\left[\frac{\partial v}{\partial t}+u\frac{\partial v}{\partial x}+v\frac{\partial v}{\partial y}+w\frac{\partial v}{\partial z}\right]=-\frac{\partial p}{\partial x}+\rho f_{y}+\mu\left[\frac{\partial^{2} v}{\partial x^{2}}+\frac{\partial^{2} v}{\partial y^{2}}+\frac{\partial^{2} v}{\partial z^{2}}\right]\right]$$

$$\rho \frac{Dw}{Dt} = -\frac{\partial p}{\partial z} + \rho f_z + \mu \left[\frac{\partial^2 w}{\partial x^2} + \frac{\partial^2 w}{\partial y^2} + \frac{\partial^2 w}{\partial z^2}\right]$$

or

<u>z</u>-direction:

$$\rho[\frac{\partial w}{\partial t} + u\frac{\partial w}{\partial x} + v\frac{\partial w}{\partial y} + w\frac{\partial w}{\partial z}] = -\frac{\partial p}{\partial z} + \rho f_z + \mu[\frac{\partial^2 w}{\partial x^2} + \frac{\partial^2 w}{\partial y^2} + \frac{\partial^2 w}{\partial z^2}]$$



### **EXPANSION OF THE NAVIER-STOKES EQUATION IN CYLINDRIC COORDINATE**

$$\rho \frac{D\vec{V}}{Dt} + \nabla P - \nabla \bullet \tilde{\tau} - \rho \, \vec{f} = 0$$

- Expansion of the Navier-Stokes Equation in Cylindrical Coordinate
- *r*-direction:

$$\begin{aligned} \frac{\partial(\rho V_{r})}{\partial t} + \frac{\partial(\rho V_{r} V_{r})}{\partial r} + \frac{\partial(\rho V_{\theta} V_{r})}{r \partial \theta} + \frac{\partial(\rho V_{Z} V_{r})}{\partial Z} + \rho \frac{V_{r} V_{r}}{r} - \rho \frac{V_{\theta} V_{\theta}}{r} \\ = -\frac{\partial p}{\partial r} + \left[\frac{\partial(r \sigma_{rr})}{r \partial r} + \frac{\partial \tau_{r\theta}}{r \partial \theta} + \frac{\partial \tau_{zr}}{\partial Z} - \frac{\sigma_{\theta\theta}}{r}\right] + \rho f_{r} \\ or \\ \frac{\partial(\rho V_{r})}{\partial t} + \frac{\partial(\rho r V_{r} V_{r})}{r \partial r} + \frac{\partial(\rho V_{\theta} V_{r})}{r \partial \theta} + \frac{\partial(\rho V_{Z} V_{r})}{\partial Z} - \rho \frac{V_{\theta} V_{\theta}}{r} \\ = -\frac{\partial p}{\partial r} + \left[\frac{\partial(r \sigma_{rr})}{r \partial r} + \frac{\partial \tau_{r\theta}}{r \partial \theta} + \frac{\partial \tau_{zr}}{\partial Z} - \frac{\sigma_{\theta\theta}}{r}\right] + \rho f_{r} \end{aligned}$$

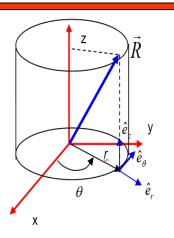
•  $\theta$ -direction:

Cylindrical coordinate system  $(r, \theta, z)$ 

$$\begin{aligned} \frac{\partial(\rho V_{\theta})}{\partial t} + \frac{\partial(\rho V_{r} V_{\theta})}{\partial r} + \frac{\partial(\rho V_{\theta} V_{\theta})}{r \partial \theta} + \frac{\partial(\rho V_{Z} V_{\theta})}{\partial Z} + \rho \frac{V_{r} V_{\theta}}{r} \\ = -\frac{1}{r} \frac{\partial p}{\partial \theta} + \left[\frac{\partial(r \tau_{r\theta})}{r \partial r} + \frac{\partial \sigma_{\theta\theta}}{r \partial \theta} + \frac{\partial \tau_{z\theta}}{\partial Z} + \frac{\tau_{\theta r}}{r}\right] + \rho f_{\theta} \\ or \\ \frac{\partial(\rho V_{\theta})}{\partial t} + \frac{\partial(\rho r V_{r} V_{\theta})}{r \partial r} + \frac{\partial(\rho V_{\theta} V_{\theta})}{r \partial \theta} + \frac{\partial(\rho V_{Z} V_{\theta})}{\partial Z} + \rho \frac{V_{r} V_{\theta}}{r} \\ = -\frac{1}{r} \frac{\partial p}{\partial \theta} + \left[\frac{\partial(r \tau_{r\theta})}{r \partial r} + \frac{\partial \sigma_{\theta\theta}}{r \partial \theta} + \frac{\partial \tau_{z\theta}}{\partial Z} + \frac{\tau_{\theta r}}{r}\right] + \rho f_{\theta} \end{aligned}$$

• *z*-direction:

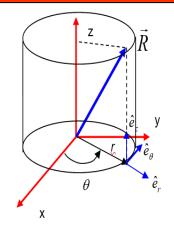
 $\frac{\partial(\rho V_{Z})}{\partial t} + \frac{\partial(\rho V_{r} V_{Z})}{\partial r} + \frac{\partial(\rho V_{\theta} V_{Z})}{r\partial \theta} + \frac{\partial(\rho V_{Z} V_{Z})}{\partial Z} = -\frac{\partial p}{\partial Z} + \left[\frac{\partial(r\tau_{rZ})}{r\partial r} + \frac{\partial\tau_{\theta Z}}{r\partial \theta} + \frac{\partial\sigma_{zz}}{\partial Z}\right] + \rho f_{Z}$ 



### **EXPANSION OF THE NAVIER-STOKES EQUATION IN CYLINDRIC COORDINATE**

$$\rho \frac{D\vec{V}}{Dt} + \nabla P - \nabla \bullet \tilde{\tau} - \rho \, \vec{f} = 0$$

• Expansion of the Navier-Stokes Equation in Cylindrical Coordinate



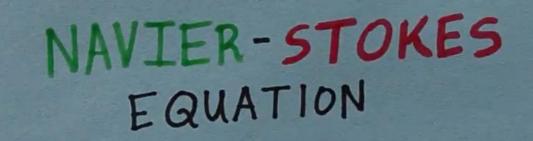
Cylindrical coordinate system  $(r, \theta, z)$ 

$$\begin{split} \widetilde{\tau} &= \mu [\nabla \vec{V} + (\nabla \vec{V})^T - \frac{2}{3} \mu (\nabla \bullet \vec{V}) \tilde{I}] \\ &= \mu \begin{bmatrix} 2 \frac{\partial V_r}{\partial r} - \frac{2}{3} (\nabla \bullet \vec{V}) & \frac{1}{r} \frac{\partial V_r}{\partial \theta} - \frac{V_{\theta}}{r} + \frac{\partial V_{\theta}}{\partial r} & \frac{\partial V_r}{\partial Z} + \frac{\partial V_z}{\partial r} \\ \frac{1}{r} \frac{\partial V_r}{\partial \theta} - \frac{V_{\theta}}{r} + \frac{\partial V_{\theta}}{\partial r} & 2(\frac{V_r}{r} + \frac{1}{r} \frac{\partial V_{\theta}}{\partial \theta}) - \frac{2}{3} (\nabla \bullet \vec{V}) & \frac{\partial V_{\theta}}{\partial Z} + \frac{1}{r} \frac{\partial V_z}{\partial \theta} \\ \frac{\partial V_z}{\partial r} + \frac{\partial V_r}{\partial Z} & \frac{1}{r} \frac{\partial V_z}{\partial \theta} + \frac{\partial V_{\theta}}{\partial Z} & \frac{\partial V_z}{\partial Z} - \frac{2}{3} (\nabla \bullet \vec{V}) \end{bmatrix} \\ &= \begin{pmatrix} \sigma_{rr} & \tau_{r\theta} & \tau_{rz} \\ \tau_{\theta r} & \sigma_{\theta \theta} & \tau_{\theta z} \end{bmatrix} \end{split}$$

 $\left( \begin{array}{ccc} au_{zr} & au_{Z heta} & au_{ZZ} \end{array} 
ight)$ 

### Navier-Stokes Equations

- Navier Stokes Equation | A Million-Dollar Question in Fluid Mechanics
  - <u>https://www.youtube.com/watch?v=XoefjJdFq6k</u>



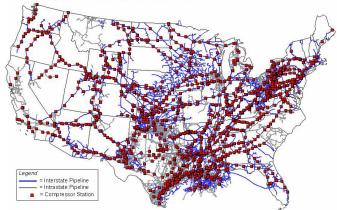
• Hagan-Poisuille flow results when the flow through a circular pipe has attained what is called a fully developed profile.

#### Assumptions:

- 1. Steady flow:  $\Rightarrow$
- 2. Incompressible flows =
- 3. Axial symmetry
- 4. In addition, we impose on velocity
- 5. No body forces.

#### $\frac{\partial}{\partial t} = 0$ $\rho, \mu \text{ are constant.}$ $\frac{\partial}{\partial \theta} = 0$ $V_r = 0$

U.S. Natural Gas Pipeline Compressor Stations Illustration, 2008







#### **Questions:**

- a). Determine the velocity profile in the pipeline.
- b). Determine the pressure drop along the pipeline.

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• Hagan-Poisuille flow results when the flow through a circular pipe has attained what is called a fully developed profile.

 $\frac{\partial}{\partial t} = 0$ 

 $\frac{\partial}{\partial \theta} = 0$ 

 $V_{r} = 0$ 

 $\rho, \mu$  are constant.

#### Assumptions:

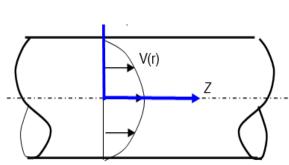
- 1. Steady flow:
- 2. Incompressible flows
- 3. Axial symmetry
- 4. In addition, we impose on velocity
- 5. No body forces.

Continuity equation:

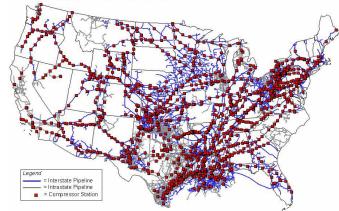
$$\frac{\partial \rho}{\partial t} + \nabla \bullet (\rho \vec{V}) = 0 \qquad \Rightarrow \qquad \nabla \bullet \vec{V} = 0$$

$$\frac{1}{r} \left[ \frac{\partial (rV_r)}{\partial r} + \frac{\partial V_{\theta}}{\partial \theta} + \frac{\partial (rV_z)}{\partial z} \right] = 0$$
  
$$\Rightarrow \frac{\partial (rV_r)}{\partial r} + \frac{\partial V_{\theta}}{\partial \theta} + \frac{\partial (rV_z)}{\partial z} = 0$$
  
$$\Rightarrow \frac{\partial (rV_z)}{\partial z} = 0 \Rightarrow \frac{\partial V_z}{\partial z} = 0$$
  
$$\Rightarrow V_z = V_z(r)$$

 $V_z$  is only the function of r, not functions of Z and  $\theta$  [due to the axial symmetry  $| \phi f \frac{\partial}{\partial \theta} = 0$ ].





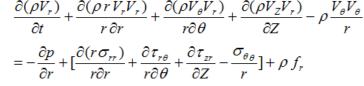


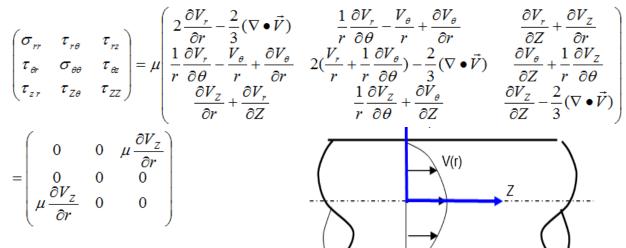




Momentum equation:

r-direction:



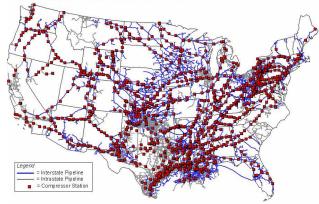


therefore

$$0 + 0 + 0 + 0 - 0 = -\frac{\partial p}{\partial r} + [0 + 0 + \frac{\partial (\mu \frac{\partial V_z}{\partial r})}{\partial Z} - 0] + 0$$
$$\Rightarrow \frac{\partial p}{\partial r} = \frac{\partial (\mu \frac{\partial V_z}{\partial r})}{\partial Z} = \mu \frac{\partial^2 V_z}{\partial Z \partial r} = \mu \frac{\partial}{\partial r} (\frac{\partial V_z}{\partial Z}) = 0$$
$$\Rightarrow \frac{\partial p}{\partial r} = 0$$

Leads to the conclusion of P = P(Z)

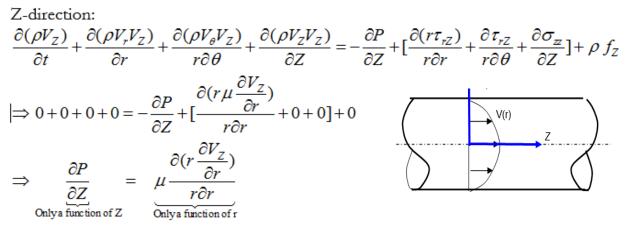
I.S. Natural Gas Pipeline Compressor Stations Illustration, 2008







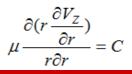
 $\begin{aligned} \theta \text{-direction:} \\ \frac{\partial(\rho V_{\theta})}{\partial t} + \frac{\partial(\rho r V_{r} V_{\theta})}{r \partial r} + \frac{\partial(\rho V_{\theta} V_{\theta})}{r \partial \theta} + \frac{\partial(\rho V_{Z} V_{\theta})}{\partial Z} + \rho \frac{V_{r} V_{\theta}}{r} \\ = -\frac{1}{r} \frac{\partial p}{\partial \theta} + \left[\frac{\partial(r \tau_{r\theta})}{r \partial r} + \frac{\partial \sigma_{\theta\theta}}{r \partial \theta} + \frac{\partial \tau_{z\theta}}{\partial Z} + \frac{\tau_{\theta r}}{r}\right] + \rho f_{\theta} \\ \Rightarrow 0 + 0 + 0 + 0 = -0 + [0 + 0 + 0 + 0] + 0 \\ \Rightarrow 0 = 0 \end{aligned}$ 



The only way the above equation is true if the two side are both equal to a constant.

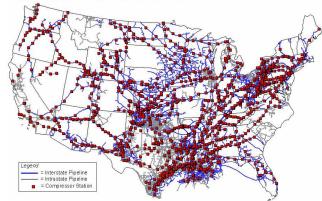
Therefore:

 $\frac{\partial p}{\partial Z} = C$ 



where C is a constant.

U.S. Natural Gas Pipeline Compressor Stations Illustration, 2008





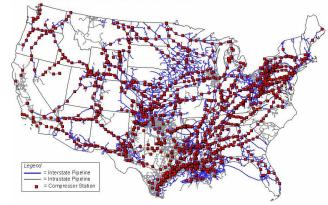


Boundary conditions:

- At the wall,  $r = r_0$ ,  $V_Z = 0$ .
- At the centerline, slope of velocity profile is zero. r = 0,  $\frac{\partial V_Z}{\partial r} = 0$

 $\mu \frac{\partial (r \frac{\partial V_Z}{\partial r})}{r \partial r} = C \quad \Rightarrow \frac{\partial (r \frac{\partial V_Z}{\partial r})}{\partial r} = \frac{Cr}{\mu}$  $\Rightarrow \int \partial (r \frac{\partial V_Z}{\partial r}) = \int \frac{Cr}{\mu} \partial r \frac{Cr}{\mu} \partial r \Rightarrow r \frac{\partial V_Z}{\partial r} = \frac{Cr^2}{2\mu} + B_1$ Since r = 0,  $\frac{\partial V_Z}{\partial r} = 0 \implies B_1 = 0$  $r\frac{\partial V_Z}{\partial r} = \frac{Cr^2}{2\mu} \implies \frac{\partial V_Z}{\partial r} = \frac{Cr}{2\mu}$  $\Rightarrow \int \partial V_Z = \int \frac{Cr}{2\mu} \partial r \Rightarrow V_Z = \frac{Cr^2}{4\mu} + B_2$ Since  $r = r_0$ ,  $V_Z = 0$   $\Rightarrow$   $B_2 = -\frac{Cr_0^2}{A_U}$ Therefore:  $V_Z = \frac{C}{4\mu} (r^2 - r_0^2)$ V(r) Since  $\frac{\partial p}{\partial z} = C$  and P = P(Z)Therefore:  $V_Z = \frac{1}{4\mu} \frac{dP}{dZ} (r^2 - r_0^2)$ 

U.S. Natural Gas Pipeline Compressor Stations Illustration, 2008



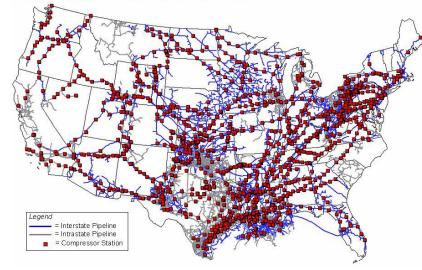


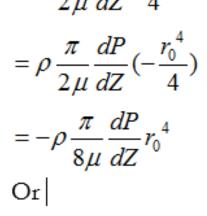


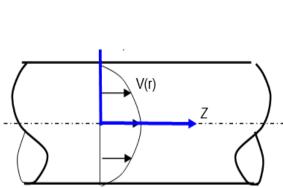
Mass flow rate:

$$\dot{m} = \int \rho V_Z dA = \int_0^{r_0} \rho \frac{1}{4\mu} \frac{dP}{dZ} (r^2 - r_0^2) (2\pi r dr)$$
$$= \rho \frac{\pi}{2\mu} \frac{dP}{dZ} (\frac{r^4}{4} - \frac{r_0^2 r^2}{2}) \Big|_0^{r_0}$$

U.S. Natural Gas Pipeline Compressor Stations Illustration, 2008







 $\frac{dP}{dZ} = -\frac{8\,\mu\dot{m}}{\rho\pi\,r_0^4}$ 

